

# Armed Services Technical Information Agency

Because of our limited supply, you are requested to return this copy WHEN IT HAS SERVED YOUR PURPOSE so that it may be made available to other requesters. Your cooperation will be appreciated.

# AD

# 30210

NOTICE: WHEN GOVERNMENT OR OTHER DRAWINGS, SPECIFICATIONS OR OTHER DATA ARE USED FOR ANY PURPOSE OTHER THAN IN CONNECTION WITH A DEFINITELY RELATED GOVERNMENT PROCUREMENT OPERATION, THE U. S. GOVERNMENT THEREBY INCURS NO RESPONSIBILITY, NOR ANY OBLIGATION WHATSOEVER; AND THE FACT THAT THE GOVERNMENT MAY HAVE FORMULATED, FURNISHED, OR IN ANY WAY SUPPLIED THE SAID DRAWINGS, SPECIFICATIONS, OR OTHER DATA IS NOT TO BE REGARDED BY IMPLICATION OR OTHERWISE AS IN ANY MANNER LICENSING THE HOLDER OR ANY OTHER PERSON OR CORPORATION, OR CONVEYING ANY RIGHTS OR PERMISSION TO MANUFACTURE OR USE OR SELL ANY PATENTED INVENTION THAT MAY IN ANY WAY BE RELATED THERETO.

Reproduced by  
DOCUMENT SERVICE CENTER  
KNOTT BUILDING, DAYTON, 2, OHIO

# UNCLASSIFIED

AD NO. 30210

ASTIA FILE COPY

# PICATINNY ARSENAL TECHNICAL DIVISION



## TECHNICAL REPORT

**SUBJECT: THE FREEZING OF MOLTEN MATERIAL IN FINITE  
REGIONS**

**ORDNANCE  
PROJECT NO. TA3-5006**

**DEPT. OF THE ARMY  
PROJECT NO. 504-10-003**

**PREPARED BY: AMINA NORDIO**

**DATE: MARCH 1954**

**P. A. SERIAL NO. 2001**

**COPY NO. 00015**

Picatinny Arsenal  
Dover, New Jersey

Technical Report No. 2001  
March 1954

THE FREEZING OF MOLTEN MATERIAL  
IN FINITE REGIONS

Ordnance Corps Project No. TA3-5006: Optimum Utilization of Explosives  
Department of Army Project No. 504-10-003

Prepared by:

*Amina Nordio*  
Amina Nordio  
Physicist

Approved:

*John D. Armitage*  
JOHN D. ARMITAGE  
Col, Ord Corps  
Chief, Tech Div  
KDG. was

## TABLE OF CONTENTS

	Page
Abstract - - - - -	1
Introduction - - - - -	2
Solidification in a Slab of Thickness <u>2a</u> - - - - -	4
The Sphere and the Infinite Cylinder - - - - -	14
The Finite Cylinder and the Parallelepiped - - - - -	19

### ABSTRACT

The analytical treatment of the problem of solidification in a slab of thickness  $2a$  is based on Lightfoot's solution for the case of a semi-infinite mass, initially at constant temperature and bounded by the plane surface  $x = 0$  kept at constant temperature. The rate of advance of the solid walls moving from the two boundary planes  $x = 0$  and  $x = 2a$ , kept at constant temperature, is given in closed form. Solidification in a sphere, in an infinite or finite cylinder, or in a parallelepiped, is obtained from the solution for the slab by correlation of temperatures due to the initial supply of heat. Numerical solutions can be calculated rapidly.

## 1. INTRODUCTION:

An analytical solution of the problem of freezing in a finite region has been attempted by Lightfoot\* who studied the rate of solidification of a medium, originally at constant temperature, and bounded by the plane surfaces  $x = 0$  and  $x = 2a$  kept at constant temperature. The problem was treated by the method of images but essential factors were neglected and the solution obtained by Lightfoot is neither complete nor exact.

An exact solution has been obtained by the same author for the case of a semi-infinite mass of molten material, originally at constant temperature, and with its bounding plane surface  $x = 0$  maintained at constant (zero) temperature. In the given solution, the diffusivity  $k$ , the specific heat  $c$ , and the density  $\rho$  of the medium, are assumed constant for all temperatures, and the same for the liquid as for the solid medium.

The solution of the problem of solidification in finite regions, given below, has been derived from Lightfoot's solution for a semi-infinite medium. The following method will give the position of the surface of separation of solid and liquid phases as a function of time. Results of this study may be of help in the investigation of a complete solution of the problem of solidification.

## 2. Solidification in a Semi-infinite Medium\*

The solution of the problem of solidification of a mass of molten material requires, in addition to known solutions, consideration of the effect on the temperature caused by the evolution of latent heat of fusion. In Lightfoot's analysis of the problem, the surface of separation of solid and liquid phases has been regarded as a moving source of heat. The speed of the plane source was determined from the condition that the temperature at the moving plane surface, at any position, is equal to the melting point of the material.

The position  $x$  of the moving plane, at time  $t$ , in a medium initially at temperature  $\theta$ , and with its bounding plane  $x = 0$  kept at zero temperature, is given by

$$x = 2\lambda\sqrt{kt} \quad (1)$$

where  $k$  is the thermal diffusivity of the medium, and the constant  $\lambda$  is obtained from

$$V = \rho\theta(\lambda) + \frac{L\sqrt{\pi}}{c} \lambda e^{-\lambda^2} \theta(\lambda) \{1 - \theta(\lambda)\} \quad (2)$$

\* Proc. London Math. Soc. (2), 31, (1930), 97

The terms used in (2) are as follows:

$v$ , the melting point of the medium

$\phi$ , the initial temperature

$L$ , the latent heat of fusion

$c$ , the specific heat

and

$$\theta(\lambda) = \frac{2}{\sqrt{\pi}} \int_0^{\lambda} e^{-\xi^2} d\xi \quad (3)$$

With this solution, the temperature  $v_1$  at a point  $x_1$ , at time  $t$ , for  $x_1 < 2\lambda\sqrt{\kappa t}$  will be:

$$v_1 = \phi \theta\left(\frac{x_1}{2\sqrt{\kappa t}}\right) + \frac{L\sqrt{\pi}}{c} \lambda e^{\lambda^2} \theta(\lambda) \left\{1 - \theta(\lambda)\right\} \quad (4)$$

and for  $x_2 > 2\lambda\sqrt{\kappa t}$

$$v_2 = \phi \theta\left(\frac{x_2}{2\sqrt{\kappa t}}\right) + \frac{L\sqrt{\pi}}{c} \lambda e^{\lambda^2} \theta(\lambda) \left\{1 - \theta\left(\frac{x_2}{2\sqrt{\kappa t}}\right)\right\} \quad (5)$$

According to (4) and (5), we notice that, while solidification proceeds from the plane  $x = 0$ , the temperatures due to the original supply of heat, and to the latent heat of fusion, can be evaluated separately.

For  $x = 2\lambda\sqrt{\kappa t}$ , at the surface of separation of solid and liquid phases, (4) and (5) coincide with (2).

The terms in (2) are constant for any position of the moving plane surface source, and can be written as:

$$v_\phi = \phi \theta(\lambda) \quad (6)$$

$$v_L = \frac{L\sqrt{\pi}}{c} \lambda e^{\lambda^2} \theta(\lambda) \left\{1 - \theta(\lambda)\right\} \quad (7)$$

$$v = v_\phi + v_L \quad (8)$$

### 3. Solidification in a Slab of Thickness $2a$

An exact solution of the problem of solidification of a slab of molten material, initially at constant temperature, can be derived from the results given in the preceding section.

For a freezing medium in the region  $2a$ , initially at temperature  $\phi$ , with the bounding plane surfaces  $x = 0$  and  $x = 2a$  kept at zero, we evaluate the sum of the temperatures due to the original supply of heat and to latent heat of fusion released on solidification by the two solid sections of the medium:  $(0, x)$  and  $(2a, 2a - x)$  as in Fig. 1.

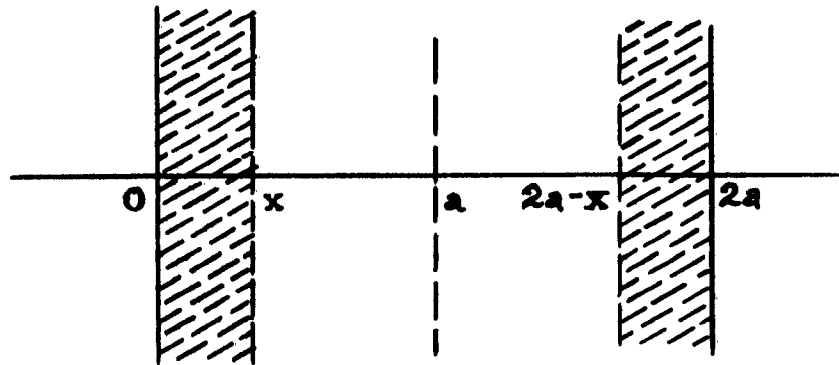


Fig. 1

The surfaces of separation of solid and liquid phases are taken as two moving plane sources of heat, and the speed of these two sources is determined from the condition that the temperature at the moving plane sources is equal to the melting point of the medium.

For the temperature at the moving sources we write the condition

$$V = V_\phi + (V_L)_1 + (V_L)_2 \quad (9)$$

where  $V_\phi$  refers to the original supply of heat, and  $(V_L)_1$ ,  $(V_L)_2$  are temperatures due to heat of fusion from solid section  $(0, x)$  and  $(2a, 2a - x)$  respectively.



It is evident that solidification will proceed from both  $x = 0$  and  $x = 2a$ , and for  $t$  small, the positions of the moving planes will be

$$x = 2\kappa_0 \sqrt{\kappa t}, \quad 2a - x = 2a - 2\kappa_0 \sqrt{\kappa t} \quad (10)$$

The constant  $\kappa_0$  of (10) is the same as for a semi-infinite medium and is given by (2). In terms of the dimensionless variable

$$\tau = \frac{\kappa}{a^2} t \quad (11)$$

forms (10) will become

$$\frac{x}{a} = 2\kappa_0 \sqrt{\tau} \quad \frac{2a-x}{a} = 2 - 2\kappa_0 \sqrt{\tau} \quad (12)$$

In the region  $2a$ , the temperature  $V_\phi$ , at any point  $x$ , at a given  $\tau$ , is the known solution

$$V_\phi = \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi x}{2a} e^{-(2n+1)^2 \pi^2 \tau/4} \quad (13)$$

During an initial period of time,  $(V_L)_1$  and  $(V_L)_2$  will be obtained from (4) and (5) as follows:

$$\text{For } \frac{x_1}{a} < 2\kappa_0 \sqrt{\tau} (V_L)_1 = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta\left(\frac{x_1}{2a\sqrt{\tau}}\right) \left\{1 - \theta(\kappa_0)\right\} \quad (14)$$

$$\text{and } (V_L)_2 = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \left\{1 - \theta\left(\frac{2a-x_1}{2a\sqrt{\tau}}\right)\right\} \quad (15)$$

$$\text{For } \frac{x_1}{a} > 2\kappa_0 \sqrt{\tau} (V_L)_1 = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \left\{1 - \theta\left(\frac{x_1}{2a\sqrt{\tau}}\right)\right\} \quad (16)$$

$$(V_L)_2 = \text{same as (15)}$$

For the moving plane source  $\frac{x}{a} = 2\kappa_0 \sqrt{\tau}$  the terms in (9) will be evaluated from:

$$V_\phi = \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi x}{2a} e^{-(2n+1)^2 \pi^2 \tau/4} \quad (17)$$

$$(V_L)_1 = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \{1 - \theta(\kappa_0)\} \quad (18)$$

$$(V_L)_2 = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \left\{1 - \theta\left(\frac{2a-x}{2a\sqrt{\tau}}\right)\right\} \quad (19)$$

Temperature  $V_\phi$ ,  $(V_L)_1$ , and  $(V_L)_2$ , at the moving plane source  $\frac{2a-x}{2a} = 2\kappa_0\sqrt{\tau}$  will be obtained with (17), (19), and (18), respectively. The sum of (17), (18), and (19), is equal to the melting point of the medium for a range of values  $0 \leq \tau \leq \tau_0$ . The upper limit  $\tau_0$  varies for each numerical case and depends upon the initial temperature  $\phi$  and the ratio  $\frac{L}{c}$ . For  $\tau$  small, the temperature at the center of the slab is given by

$$v_{x=a} = \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi}{2} e^{-(2n+1)^2 \pi^2 \tau/4} + 2 \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \left\{1 - \theta\left(\frac{1}{2\sqrt{\tau}}\right)\right\} \quad (20)$$

and we can verify that, when

$$\begin{aligned} \tau \leq \tau_0 & \quad v_{(x=a)} = \phi \\ \tau > \tau_0 & \quad v_{(x=a)} < \phi \end{aligned}$$

In each numerical case, therefore, the upper limit  $\tau_0$  can be obtained with (20). We see also that, during the entire period of time  $\tau_0$ , the boundary conditions of the problem will be satisfied.

In the case of a semi-infinite medium and  $\frac{x}{2a} > 2\kappa_0\sqrt{\tau}$ , or  $\frac{x}{2a} > 2\kappa_0\sqrt{\tau}$ , the temperature given by (5) is the sum of the two terms

$$V_\phi = \phi \theta\left(\frac{x}{2a\sqrt{\tau}}\right) \quad (21)$$

$$V_L = \frac{L\sqrt{\pi}}{c} \kappa_0 e^{\kappa_0^2} \theta(\kappa_0) \left\{1 - \theta\left(\frac{x}{2a\sqrt{\tau}}\right)\right\} \quad (22)$$

Solution (21) is correct for large values of  $\tau$  but, for small values,  $V_\phi$  must be equal to  $\phi$  at a finite distance from the boundary  $x = 0$ . Where  $V_\phi = \phi$ , we will obtain  $(V_L) = 0$ . In general, the boundary condition is satisfied when the temperature at the center of a slab of thickness  $4a$ , initially at  $\phi$  and with bounding surfaces kept at zero is

$$(V_\phi)_{\substack{x=2a \\ \tau=\tau_0}} = \phi = \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi}{2} e^{-(2n+1)^2 \pi^2 \tau_0 / 16} \quad (23)$$

This condition can be verified for any  $\tau_0$ , including the limiting case where

$$\frac{x_0}{a} = 2\epsilon_0 \sqrt{\tau_0} = 1$$

and we can write

$$(V_L)_{\substack{x=2a \\ \tau=\tau_0}} = 0$$

During the period of time  $\tau_0$ , the two plane sources move from the boundaries toward the center of the slab as in semi-infinite medium. The temperature distributions  $V_\phi$ ,  $(V_L)_1$ , and  $(V_L)_2$ , and their sum, computed for a numerical example, with  $\tau = \tau_0$ , are shown on Fig. 2.

In the region  $2a$ , initially and till solidification is complete, the correct mathematical forms of  $(V_L)_1$  and  $(V_L)_2$ , at the plane source moving from  $x = 0$  to  $x = \frac{1}{2}a$  are:

$$(V_L)_1 = \frac{L}{ac} \sum_{n=1}^{\infty} \sin \frac{n\pi x}{2a} \int_0^\tau \sin \frac{n\pi x'}{2a} e^{-n^2 \pi^2 (\frac{\tau-\tau'}{4})} \frac{dx'}{d\tau} d\tau' \quad (24)$$

$$(V_L)_2 = \frac{L}{ac} \sum_{n=1}^{\infty} \sin \frac{n\pi x}{2a} \int_0^\tau \sin \frac{n\pi(2a-x')}{2a} e^{-n^2 \pi^2 (\frac{\tau-\tau'}{4})} \frac{dx'}{d\tau} d\tau' \quad (25)$$

During the interval  $0 \leq \tau \leq \tau_0$ ,  $(V_L)_1$  is a constant obtained from (18) while  $(V_L)_2$ , evaluated from (19), is initially zero but may assume increasing values as  $\tau$  approaches  $\tau_0$ . However, during the entire interval  $0 \leq \tau \leq \tau_0$

$$(V_L)_2 = V - \{V_\phi + (V_L)_1\} \quad (26)$$

The rate of solidification in the range  $0 \leq x \leq x_0$  might be obtained directly from (24) and (25) if the series of integrals in the above forms could be evaluated exactly. So far, available graphical or numerical techniques are unsatisfactory.

Lightfoot shows that a rate of solidification can be obtained by trial method, and assumes a position of the moving planes given by

$$\frac{x}{a} = b - 2K\sqrt{\tau_2 - \tau} \quad \begin{matrix} \tau \geq \tau_0 \\ x \geq x_0 \end{matrix}$$

where  $b$ ,  $K$ , and  $\tau_2$ , are chosen for continuity and satisfy the conditions

$$\frac{b}{a} = 2K\sqrt{\tau_0} \left\{ 1 + \frac{K^2}{\tau_0^2} \right\} \quad \tau_2 \tau_0 \left\{ 1 + \frac{K^2}{\tau_0^2} \right\}$$

The constant  $K$  remains undetermined and its value must be adjusted for correct temperature at prescribed time. According to the author, this method requires "great labour".

For solidification in the range  $x_0 \leq x \leq a$  we assume a position of the moving plane given by

$$\frac{x}{a} = -b + 2k_2\sqrt{\tau - \tau_2} \quad \begin{matrix} x \geq x_0 \\ \tau \geq \tau_0 \end{matrix} \quad (27)$$

where  $b$ ,  $k_2$ , and  $\tau_2$ , are to be determined from the condition that the temperature at the moving plane is equal to the melting point of the medium. For continuity of the two functions of  $\tau$ , (11) and (27) and their first derivatives at  $x = x_0$ , we make

$$\tau_2 = \tau_0 \left( 1 - \frac{k_2^2}{\tau_0^2} \right) \quad (28)$$

We determine  $b$  and  $t_a$  from the following:

$$\begin{aligned} \frac{x_2}{a} &= -b + 2 t_a \sqrt{\tau_2 - \tau_2} \\ 1 &= -b + 2 t_a \sqrt{\tau_2 - \tau_2} \end{aligned} \quad (29)$$

The second equation of (29) refers to complete solidification at a definite  $\tau_2$ , to be obtained according to the following considerations.

While the source of heat moves from  $x = 0$  to  $x = x_0$ , temperature  $(V_L)_1$  is obtained from (18), and is a constant at any point in this range. At the same plane source,  $(V_L)_2$  will vary from zero to a known value. Solidification is complete when the source moving from  $x = 0$ , and the source moving from  $x = 2a$ , merge at  $x = a$ . At this position we will obtain  $(V_L)_1 = (V_L)_2$  regardless of their value at any other point.

As the plane source moves from  $x = x_0$  to  $x = a$ , temperature  $(V_L)_1$  may be:

- (a) continuously increasing
- (b) continuously decreasing
- (c) constant =  $\frac{L\sqrt{\pi}}{c} t_0 e^{t_0^2} \theta(t_0) \{1 - \theta(t_0)\}$

The rate of solidification for (c) can be easily obtained, as the temperature, due to the original supply of heat at  $x = a$  and  $\tau = \tau_2$  will be given by

$$\begin{aligned} V_2 &= \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi}{2} e^{-(2n+1)^2 \pi^2 \tau_2 / 4} \\ &= V - \frac{2L\sqrt{\pi}}{c} t_0 e^{t_0^2} \theta(t_0) \{1 - \theta(t_0)\} \end{aligned} \quad (30)$$

The value  $\tau_2$  of (30) may be found graphically and checked numerically.

The functions of  $\tau$ , corresponding to (a), (b), and (c), will be obtained by substituting in the second equations of (29) a value  $\tau_2$

- (a) greater than in (30)
- (b) smaller than in (30)
- (c) as in (30)

and may be represented as in Fig. 3.

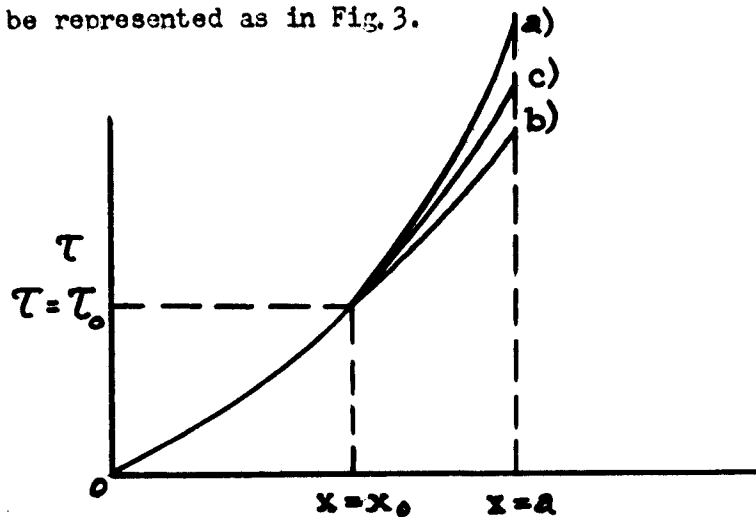


Fig. 3

The speed of the moving planes for (a), (b), and (c), will be given by

$$\frac{dx}{d\tau} = \frac{k_2}{\sqrt{\tau - \tau_2}} \quad \begin{matrix} x_0 \leq x \leq a \\ \tau_0 \leq \tau \leq \tau_2 \end{matrix} \quad (31)$$

and for each case will depend on  $k_2$  and  $\tau_2$ .

In any numerical example, the lowest rate of solidification will be obtained for case (a) and the highest rate for case (b).

The rate of solidification is given also by the rate of conduction of heat at the source \*, or

$$K\rho c \left\{ \frac{d(V_L)_1}{dx_{\text{solid}}} - \frac{d(V_L)_1}{dx_{\text{molten}}} \right\} \quad (32)$$

\* Lightfoot, loc. cit.

Graphically, the rate of conduction for case (a) may be represented as in Fig. 4. A large increase of  $(V_L)_i$  implies an increase in the rate of conduction at the source, or rate of solidification. The rate computed with (31) will follow an opposite trend. The difference between graphical and computed rates tends to zero when  $(V_L)_{x=a} \rightarrow (V_L)_{x=x_0}$ .

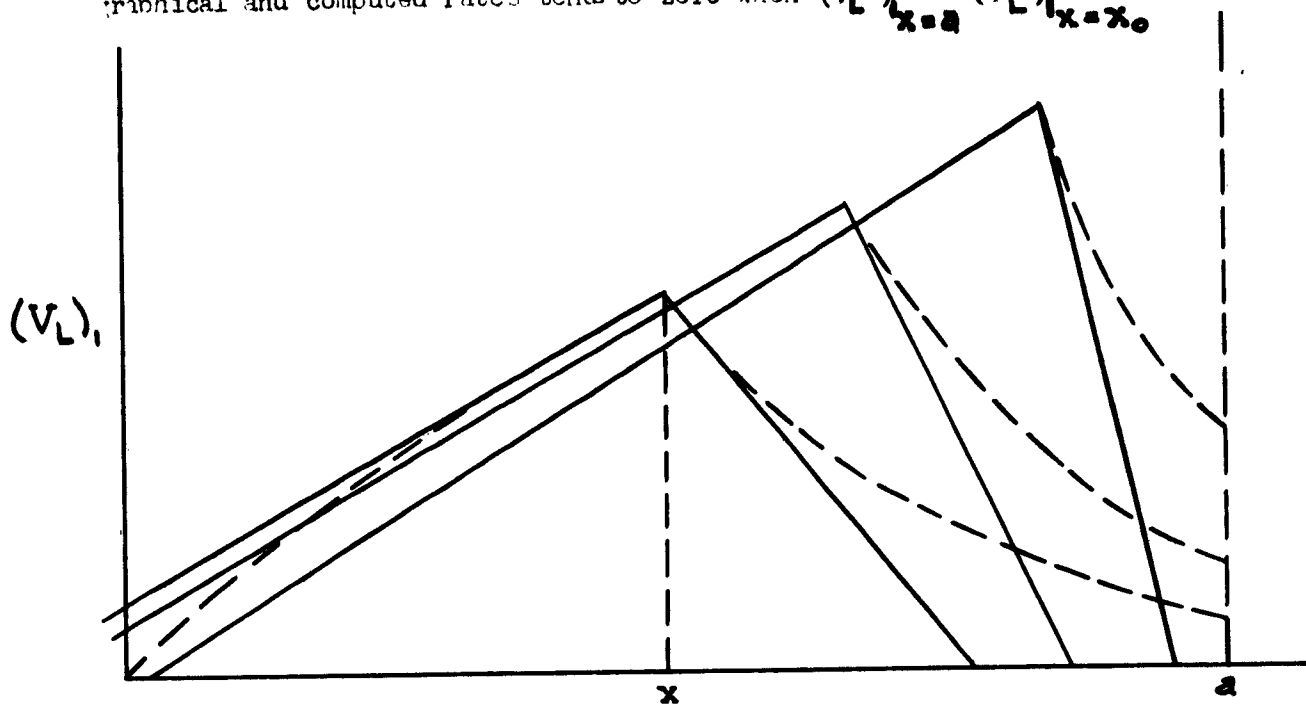


Fig. 4\*

Case (b), represented in Fig. 5, shows that if  $(V_L)_i$  decreases, the graphical rate will be lower than the computed rate. Their difference tends to zero when  $(V_L)_{x=a} \rightarrow (V_L)_{x=x_0}$ .

\*Dotted lines on Fig. 4, 5, and 6 are temperature distributions  $(V_L)_i$ . Solid lines are drawn tangent to distributions at the sources.

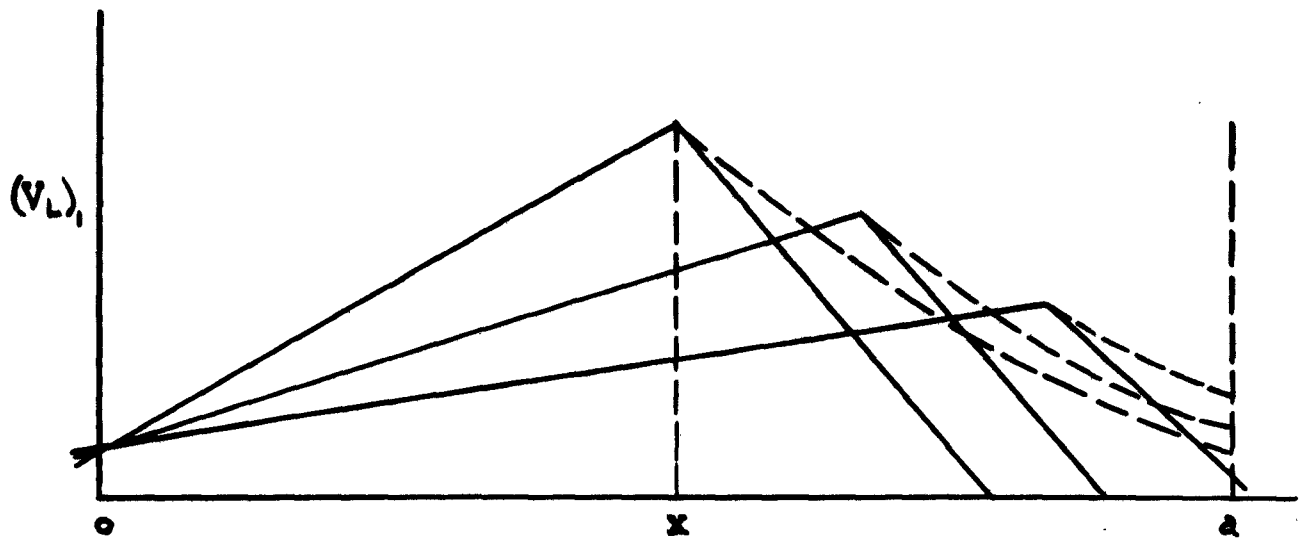


Fig. 5

In case (c), Figure 6, graphical and computed rates will coincide. As the moving planes  $\underline{x}$  and  $\underline{2a} - \underline{x}$  approach  $\underline{x} = \underline{a}$ , the rate of solidification decreases and tends to a constant. A constant temperature  $(V_L)_i$ , at any position of the plane source moving from  $\underline{x} = 0$  (or  $(V_L)_i$  at the plane source moving from  $\underline{x} = \underline{2a}$ ) is the correct assumption.

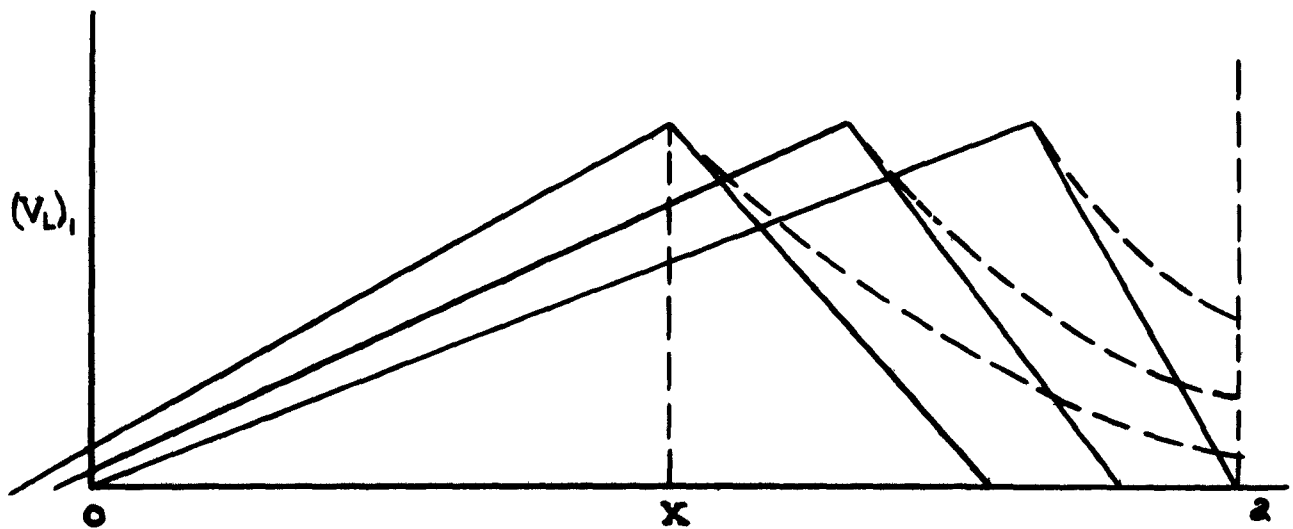


Fig. 6



We may state that in a freezing medium, initially at temperature  $\phi$ , and with its bounding surfaces  $x = 0$ , and  $x = 2a$ , kept at zero, the positions of the surfaces of separation of solid and liquid phases are given by the two functions of  $\tau$

$$\begin{aligned} \frac{x}{a} &= 2\lambda_0 \sqrt{\tau} & 0 \leq \tau \leq \tau_0 \\ \frac{x}{a} &= -b + 2\lambda_2 \sqrt{\tau - \tau_0} & \tau_0 \leq \tau \leq \tau_2 \end{aligned} \quad (33)$$

The constants and the ranges of  $\tau$  of (33) will vary for each numerical case and are to be determined from the data of the problem according to the conditions given in this section.

In the forms of (33), the thermal diffusivity  $k$ , included in the dimensionless variable  $\tau = \frac{k}{V^2} \tau$ , is not undetermined. We assume  $k$  constant and equal to the average diffusivity of the medium in the temperature range  $V$  to  $\phi$ , i.e. in the molten material. This constant is to be determined experimentally, and methods of measurement may be based on solutions given in the following sections.

The complete exact solution of the problem, for the special case  $\phi = V$  (i.e. initial temperature is at melting point of the medium) is given by Lightfoot. In this case solidification proceeds from each boundary  $x = 0$  and  $x = 2a$ , independently, and the position of the moving plane  $x$  (or  $2a - x$ ), until solidification is complete, is given by

$$\frac{x}{a} = 2\lambda \sqrt{\tau} \quad (34)$$

where  $\lambda$  is obtained from (2) as for a semi-infinite medium.

In any numerical example  $\phi = V$ , we can write

$$\tau_2 = \left(\frac{1}{2\lambda}\right)^2 \quad (35)$$

and we verify that for  $\tau_2$  as in (35) we will obtain

$$V_{\phi} = \frac{4V}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi}{2} e^{-(2n+1)^2 \pi^2 \tau_{2/4}}$$

$$= V - \frac{2L\sqrt{\pi}}{c} \sqrt{t} e^{\sqrt{\pi} t} \theta(\sqrt{\pi} t) \{1 - \theta(\sqrt{\pi} t)\} \quad (36)$$

Some numerical solutions for media having the same melting point are shown in Figure 7. The initial temperatures  $\phi$  and ratios  $\frac{L}{c}$  are adjusted for equal value of constant  $k_0$ .

#### 4. The Sphere and the Infinite Cylinder

In the solution of the problem of solidification in the finite region  $2a$ , the planes of separation of solid and liquid phases have been regarded as moving plane sources of heat. Their speed was obtained from the condition that the temperature at the sources is equal to the melting point of the medium. The temperature due to the original supply of heat is evaluated from a known solution while the temperature due to latent heat of fusion is determined from analysis. Solidification in a sphere, or in a cylinder, gives rise to spherical, or cylindrical surface sources of heat, and the forms of  $V_L$ , for these cases, are similar to (24) or (25). As for moving plane surfaces, correct evaluation of the series of integrals can not be obtained by available methods. The temperature ( $V_L$ ), at the surfaces of separation of solid and liquid phases will be determined from analysis.

The known solutions  $V_{\phi}$  in the slab of thickness  $2a$ , in the infinite cylinder  $a > r \geq 0$ , and in the sphere  $a > r \geq 0$ , initially at  $\phi$  and with boundary surfaces kept at zero, expressed in terms of the dimensionless variable  $\tau = \frac{k}{a^2} t$ , are as follows:

$$(V_\phi)_1 = \frac{4\phi}{\pi} \sum_{n=0}^{\infty} \frac{1}{2n+1} \sin \frac{(2n+1)\pi x}{2a} e^{-\frac{(2n+1)^2 \pi^2 \tau}{4}} \quad (37)$$

$$(V_\phi)_2 = 2\phi \sum_{n=0}^{\infty} \frac{J_0(r\alpha_n)}{a_n J_1(a\alpha_n)} e^{-a_n^2 \tau_2} \quad (38)$$

$$(V_\phi)_3 = \frac{2a\phi}{\pi r} \sum_{n=0}^{\infty} \frac{(-1)^n}{n+1} \sin \frac{(n+1)\pi r}{a} e^{-(n+1)^2 \pi^2 \tau_3} \quad (39)$$

where the subscripts 1, 2, and 3, refer to the slab, the cylinder and the sphere, in that order.

The temperature distributions obtained with (37), (38), and (39), for the same medium, with dimension  $a$  and initial temperature  $\phi$ , the same for the three regions, will never result as in Figure 8, regardless of  $\tau_1$ ,  $\tau_2$ , and  $\tau_3$ . Identical distributions in any given range  $x = a - r$ ,  $2a - x = a - r$  and  $0 < x \leq a$ , may be obtained for particular values of  $\tau_1$ ,  $\tau_2$ , and  $\tau_3$ , as in Figure 9.

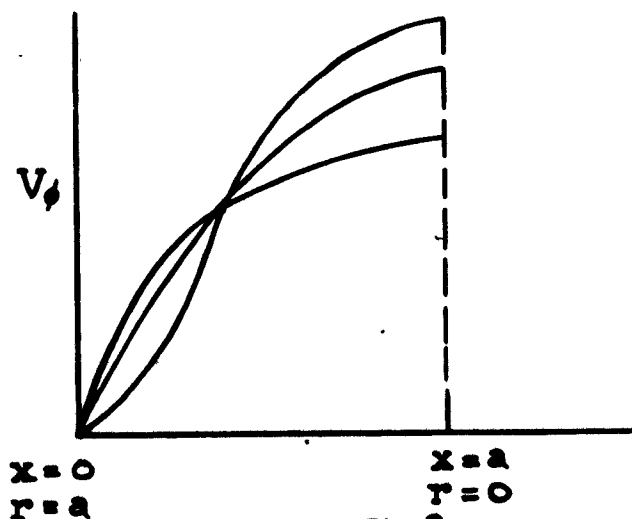


Fig. 8

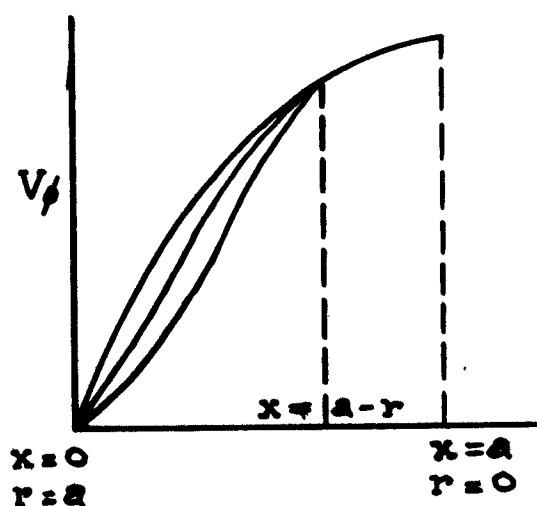


Fig. 9

In the problem of solidification we assume that, in all three regions, the surfaces of separation of solid and liquid phases are at equal distance from the boundary (we may have  $\tau_1 = \tau_2 = \tau_3$ ) and for these surfaces we write

$$V_\phi + V_L = V \quad (40)$$

At these surfaces we obtain also

$$\begin{aligned} (V_\phi)_1 &= (V_\phi)_2 = (V_\phi)_3 \\ (V_L)_1 &= (V_L)_2 = (V_L)_3 \end{aligned} \quad (41)$$

where  $(V_L)_1$ , in this instance, is the temperature caused by all the heat of fusion flowing in the slab 2a.

We will prove briefly that (41) is a necessary condition in the solution of these problems.

We consider the slab and the infinite cylinder, solid at unit distance from the boundary, or at

$$x = a - r = 1 \quad (42)$$

and at the surfaces of separation, we make

$$(V_\phi)_1 > (V_\phi)_2 \quad (43)$$

The inequality of (43) means that, in the solid part of the slab, the temperature distribution  $(V_\phi)_1$  would be higher, at all points, than the temperature distributions  $(V_\phi)_2$  in the solid part of the cylinder. During solidification of unit volumes of medium adjacent to the external

surfaces, the total flow of original heat, per unit area, would be greater at the cylindrical surface  $x = R$  than at the corresponding plane surface  $x = a$ . During solidification, in any region, original heat and latent heat of fusion flow to the boundary concurrently and are subject to the same boundary conditions. With a greater total flow of original heat we would obtain also a greater total flow of latent heat of fusion; the temperature distribution ( $V_L$ ), in the solid part of the slab would be higher, at all points, than the temperature distribution ( $V_L$ ) in the cylinder.

Assumption (43) leads to the conclusion that, either in the slab or the cylinder, the temperature at the surface of separation of solid and liquid phases would result

$$V_s + V_L \neq V$$

(44)

The same conclusion will be reached reversing the inequality in (44) and for subsequent elements of volume, until solidification is complete. The same result will be reached comparing solidification in a slab and in a sphere.

This conclusion opens the way to the solution of the problems of solidification previously discussed and appears to have been overlooked in past work on the subject.

Since the solution of the problems of solidification for the slab, the infinite cylinder, and the sphere, must satisfy condition (41), the solutions for the cylinder and the sphere can be derived from the solution obtained for the slab.

For solidification in the slab, at a given distance  $x/a$ , we evaluate  $T$  from the two forms of (33) and solve (37) for these values of  $T$  and  $x/a$ . In (38) and (39) we substitute  $x/a$  and  $V_s$  as obtained from (37). The values  $T_s$  and  $T_l$  given by the (38) and (39) can be obtained graphically and checked numerically.

Figure 10 shows that, initially, the rates of solidification in all three regions coincide. After initial periods, the rates of solidification of a cylinder and a sphere will assume constant values; in a sphere more rapidly than in a cylinder.

5. The Slab, the Infinite Cylinder, and the Sphere, Initial temperature constant and radiation at the surface into a medium at constant temperature

As in the previous cases, the solutions of these problems are derived from the solution obtained for solidification in a slab, with initial temperature constant and bounding surfaces kept at zero.

In a medium, initially at  $\phi$  and with radiation at  $x = -a$  and  $x = +a$  into a medium at zero, the temperature distribution  $V_\phi$  is given by

$$V_\phi = \phi \sum_{n=1}^{\infty} \frac{2A \cos \frac{\beta_n x}{a}}{[\beta_n + A + A^2] \cos \beta_n} e^{-\beta_n^2 \tau} \quad (45)*$$

where  $A = \frac{ah}{k}$ ,  $h$  being the heat transfer coefficient of the surface, and  $\beta_1, \beta_2, \dots, \beta_n$  the positive roots of

$$\beta \tan \beta = a h \quad (46)$$

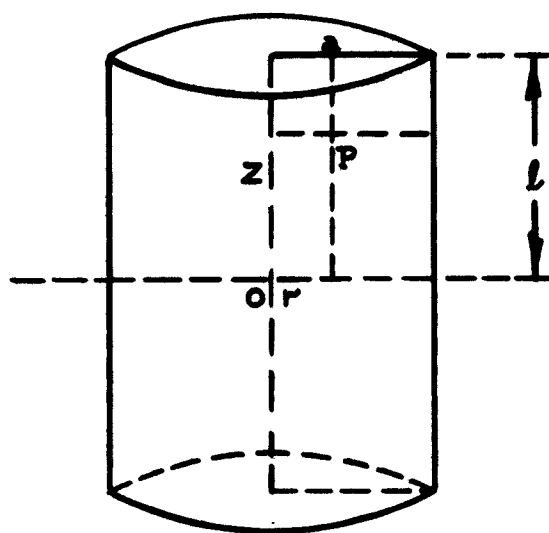
The temperature distribution obtained with (46) for any value of  $n$  and  $\tau$  and the temperature distribution obtained from (37) for the slab, for any value of  $\tau$ , will never cross each other, as in Figure 8. The difference between temperature gradients at corresponding points, through the entire region, will always be  $\geq 0$ , or always  $\leq 0$ . We conclude that, when in the region  $-a < x < +a$ , at constant initial temperature and radiating at  $x = -a$  and at  $x = +a$  into a medium at constant temperature, we find the surfaces of separation of solid and liquid phases at the same distance from the boundary as in the slab of (37), condition (41) will be satisfied. The solution of the problem for this case is obtained by the method outlined in section 4.

Condition (41) applies also to an infinite cylinder and to a sphere, initially at constant temperature and cooled by radiation into a medium at constant temperature. The two problems can be solved by the same method and forms of  $V_\phi$  for these cases can be found in textbooks.

\* Carslaw and Jaeger, Conduction of Heat In Solids, p. 100

## 6. The Finite Cylinder and the Parallelepiped

In the cylinder of Figure 11 initially at constant temperature  $\phi$  and with boundary surfaces kept at zero, the surface of separation of solid and liquid phases will be at a point



$$P: \begin{cases} \frac{l-z}{l} = \frac{1}{4} \\ \frac{a-r}{a} = \frac{2}{3} \end{cases} \quad (47)$$

Fig 11

when  $(V_{\phi})$  is equal to the temperature  $V_{\phi}$  obtained in the slab of (37) at the surface of separation  $\frac{z}{l} = \frac{1}{4}$ . In general, solidification in a finite cylinder occurs with the higher of the two temperatures  $V_{\phi}$  obtained for the slab at the two positions given by the coordinates of the point as in (47).

The known solution  $V_{\phi}$  in a finite cylinder, initially at  $\phi$  and with boundary surfaces kept at zero, expressed in terms of the dimensionless variable  $\tau = \frac{t}{a^2} \tau$ , is as follows:

$$V_{\phi} = \frac{\phi_0}{\pi} \sum_{n=1}^{\infty} \frac{J_0(zd_n)}{a_n J_1(ad_n)} e^{-a_n^2 \tau} \cdot \sum_{m=0}^{\infty} \frac{(-1)^m}{2m+1} \cos \frac{(2m+1)\pi z}{2l} e^{-(2m+1)^2 \pi^2 \left(\frac{z}{2l}\right)^2 \tau}$$

(48)

Solution  $V_{\phi}$  for a finite cylinder, initially at  $\phi$  and cooled by radiation into a medium at constant temperature is given in textbooks.

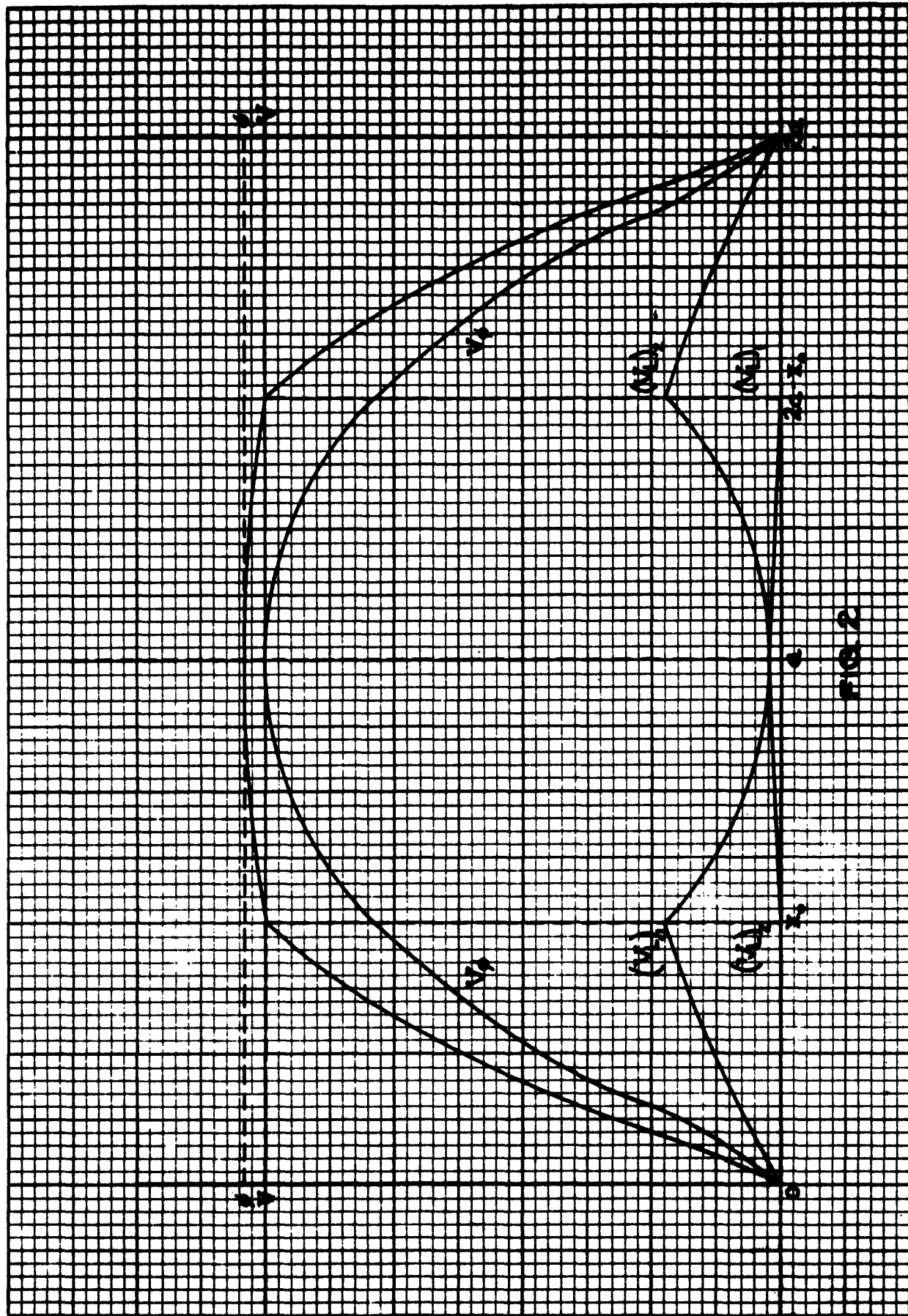
Solidification in parallelepiped is solved by the same method.

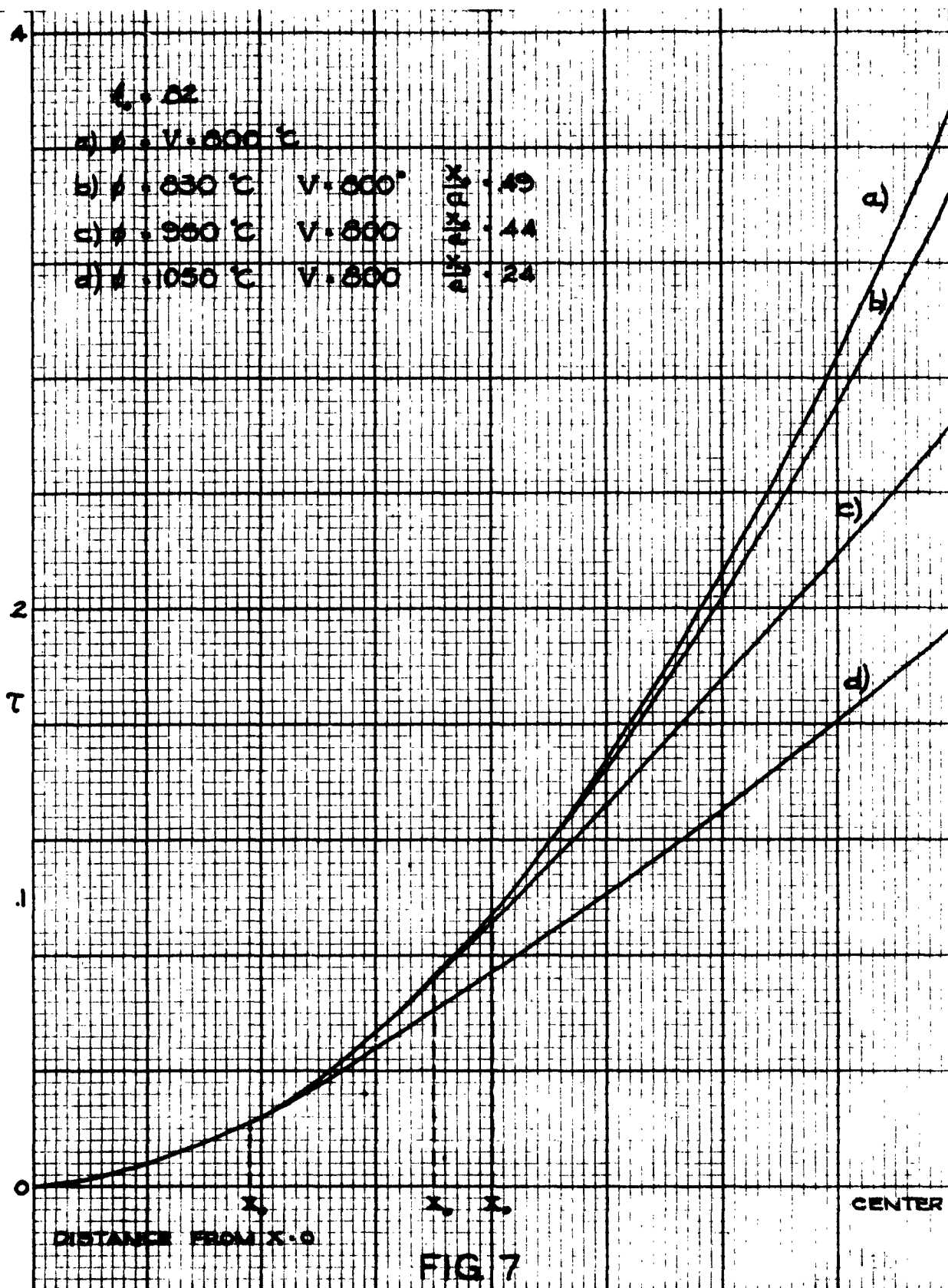
Figure 12 shows surfaces of separation in a finite cylinder. The numerical data are the same as for (b) of Figure 7.

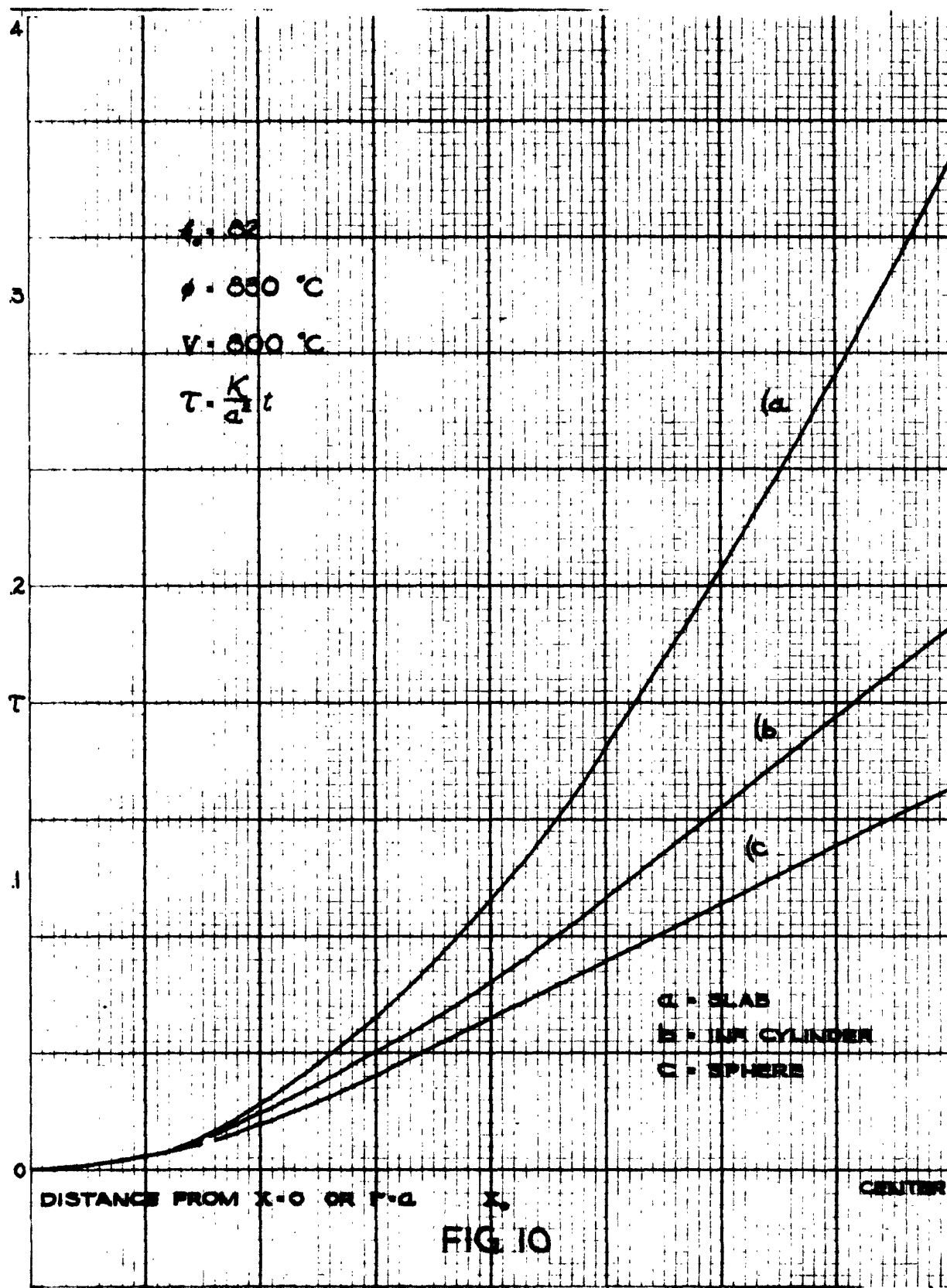
Inclousures:

1. Figure 2
2. Figure 7
3. Figure 10
4. Figure 12









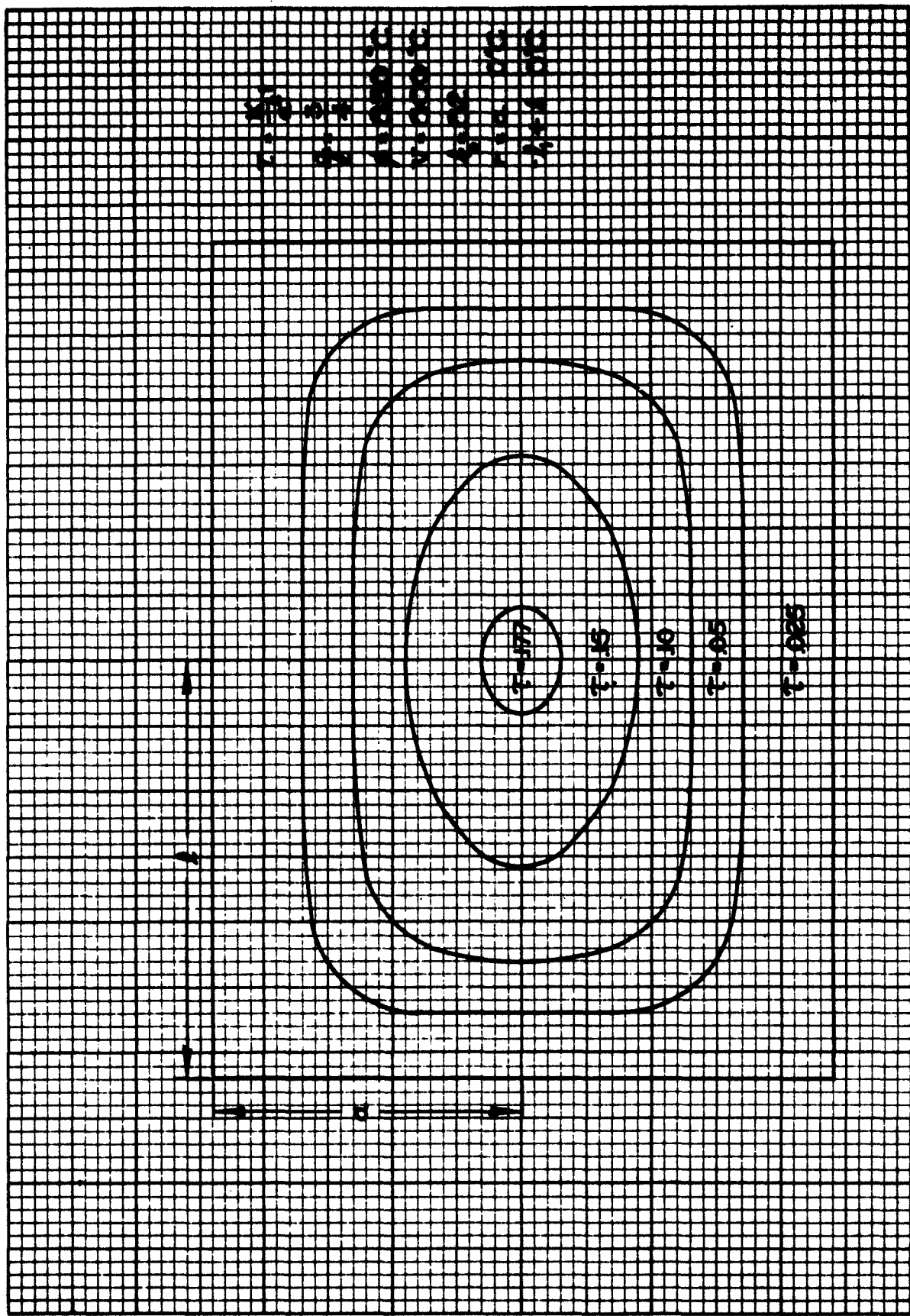


FIG. 12

# DISTRIBUTION LIST

Copy No.

Picatinny Arsenal  
Dover, New Jersey  
ATTN: Technical Library

1-2

Chief of Ordnance  
Department of the Army  
Washington 25, D. C.  
ATTN: ORDTA  
ORDTX-AR

3  
4

Commanding Officer  
Frankford Arsenal  
Bridge & Tacony Streets  
Philadelphia 37, Pa.  
ATTN: Pitman-Dunn Laboratory

5

Commanding General  
Aberdeen Proving Ground  
Maryland

6

Commander  
U. S. Naval Ordnance Laboratory  
White Oak, Silver Spring, Maryland  
ATTN: Library

7

Department of the Navy  
Bureau of Ordnance  
Washington 25, D. C.

8-9

Commander  
U. S. Naval Ordnance Test Station  
Inyokern, China Lake, California  
ATTN: Technical Library Branch

10

Commanding General  
Redstone Arsenal  
Huntsville, Alabama

11

Director  
Los Alamos Scientific Laboratory  
ATTN: Dr. D. P. Mac Dougall

12

*Los Alamos, New Mexico*

Armed Services Technical Information Agency  
Document Service Center  
Knott Building  
Dayton 2, Ohio  
ATTN: DSC-SD

Copy No.

13-17

Commanding General  
Ordnance Ammunition Center  
Joliet, Illinois  
ATTN: ORDLY-R

18